

# MECHANICAL ADVANTAGES OF A TRUSS-STRUCTURE-BASED FRACTURE FIXATION SYSTEM – A NOVEL FRACTURE FIXATION DEVICE “PINFIX” –

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## ABSTRACT

A small, light, ball-joint device called PinFix, which can instantly convert a simple percutaneous cross pin fracture fixation system into a rigid external fracture fixation system based on truss structure, was developed. The purpose of this study was to compare the mechanical load and breaking strength of this truss-structure-based fixation system to that of the conventionally used external cantilever structure-based fixation system. Three types of mechanical loading tests, axial, bending, and torsion, were performed on an artificial fractured bone treated with either three-dimensional PinFix fixation, two-dimensional PinFix fixation, or conventional external fixation. The three- and two-dimensional PinFix fixations showed significantly more stiffness than conventional fixation on all three loading tests. Finite element analysis was next performed to calculate the stress distribution of the parts in PinFix and in the conventional fixator. The applied stress to the rod and connectors of PinFix was much less than that of the conventional external fixator. These results reflected the physical characteristic of truss structure in which applied load is converted to pure tension or compression forces along the members of the PinFix. In conclusion, PinFix is a simple fracture fixation system that has a truss-structure with a high rigidity.

Key Words: truss-structure, fracture fixation system, PinFix

## INTRODUCTION

Providing sufficient stability, preservation of circulation, and avoidance of infection at the fracture site are the three most important requirements for promoting fracture healing.<sup>1-3)</sup> In order to provide an optimal healing environment, various fracture fixation techniques have been developed, such as transcutaneous Kirschner wire (K-wire) fixation, external fixation, plating, and intramedullary nailing.<sup>3-5)</sup> Each method has its advantages and disadvantages. In terms of circulation preservation and infection avoidance, transcutaneous Kirschner wire (K-wire) fixation and external fixation have theoretical advantages over plating and intramedullary nailing due to their minimum damage to soft tissues and fracture fragments.<sup>6, 7)</sup> In addition, these procedures are usually performed following closed reduction. Conversely, plating and intramedullary nailing tend

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to be more harmful to the fracture site, but they generally allow more precise reduction, provide better stability, and allow patients more freedom of daily activity during fracture healing.<sup>8-10)</sup>

Percutaneous K-wire fixation is a cost-effective procedure that does not require any special devices for implementation. It is also technically less demanding and is highly versatile in terms of its range of application.<sup>11)</sup> However, fixation provided by K-wires alone is less secure and carries significantly higher risk of loss of reduction than other techniques even when applied to fractures in the upper extremities.<sup>12)</sup> External fixation can generally provide much higher stability at the fracture site than K-wire fixation.<sup>13-15)</sup> Connections between the external fixation device and its screws are located outside of the body. Because of the much longer lever arms compared to those of internal fixation devices, a huge moment of force develops around them. Therefore, all of the components have to be rigidly fixed so that the fixator system can keep its shape. This makes the external fixator a heavy, cumbersome, and less versatile device. As a result, most external fixators are designed for a specific site or a specific type of fracture and are supplied with special jigs for assembly. External fixator devices use a cantilever structure, and they are only supported on one side with screws that project horizontally in space.<sup>16)</sup> These factors make the external fixation devices more expensive and more technically demanding compared to K-wire fixation.

In architecture, bridges are designed and constructed using cantilever methods. To ameliorate the moment around the connections and to make them more robust against cyclic drifts, the use of diagonal support frames and sway braces is highly recommended.<sup>17)</sup> The X-bracing system is one of the easiest methods to transfer lateral loads in buildings.<sup>18)</sup> Cross-bracing systems, with or without friction dampers, are believed to be fundamental for the seismic response.<sup>19)</sup> Therefore, cantilevered bridges in architecture are seldom completed as true cantilevers but are instead completed as truss bridges.<sup>17)</sup> A truss is a structure comprising one or more triangular units constructed with straight members whose ends are connected at joints, referred to as nodes,<sup>20)</sup> and external forces act only at the nodes and result in forces on the members that are either tensile or compressive, resulting in exclusion of moment.<sup>21, 22)</sup>

We have developed a novel ball-joint device named the PinFix that instantly converts a simple crossed K-wire fixation system into a robust external fixation system by constructing a truss structure.

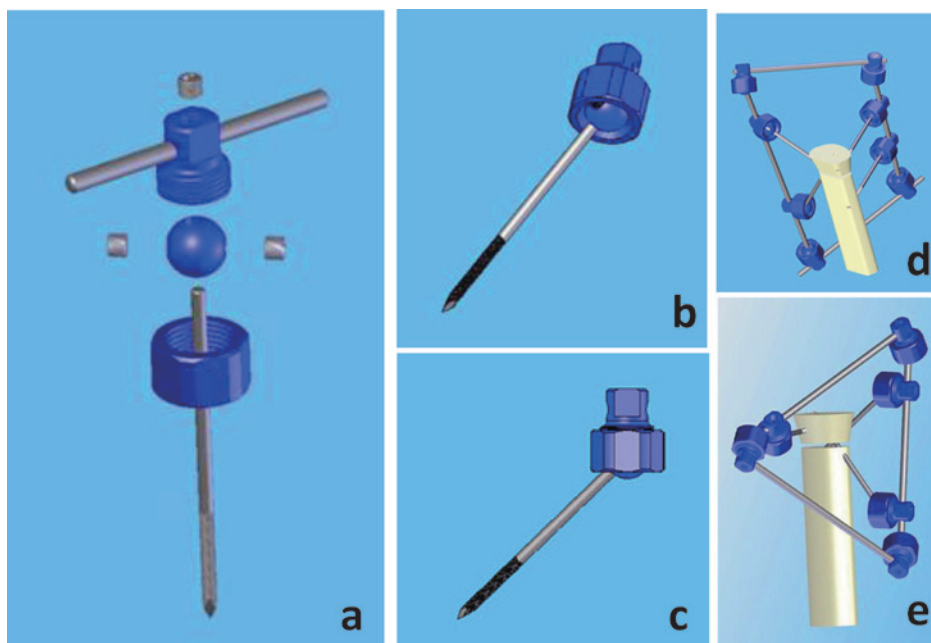
The purpose of this study was to compare the mechanical load and breaking strength of the fracture fixation of PinFix to those of the conventional cantilever external fixator using an artificial bone model and finite element analysis.

## MATERIALS AND METHODS

### *Mechanical loading tests*

The PinFix is a plastic ball-joint weighing 2.9 g that can connect pins ( $\phi$ 1.6–2.4 mm) with rods ( $\phi$ 3 mm) at any desired angle to form a truss-structure-based fracture fixation system. As shown in Fig. 1(a-e), the pins are inserted in a crisscross fashion across the fracture site. This is a universal system that can be used to construct either two- or three-dimensional fixations in any configuration.

A 30 radius sawbone with a cancellous inner core and a foam cortical shell (26 cm long, 5.5-mm canal diameter; Model #1027, Pacific Research Laboratories, Inc., Vashon, WA, USA) was prepared for axial and bending load testing. The proximal radial shaft of each specimen was potted with a metallic adapter, leaving approximately 10 cm of the radius exposed. For the axial and bending load testing, an oscillating saw was used to create a transverse osteotomy

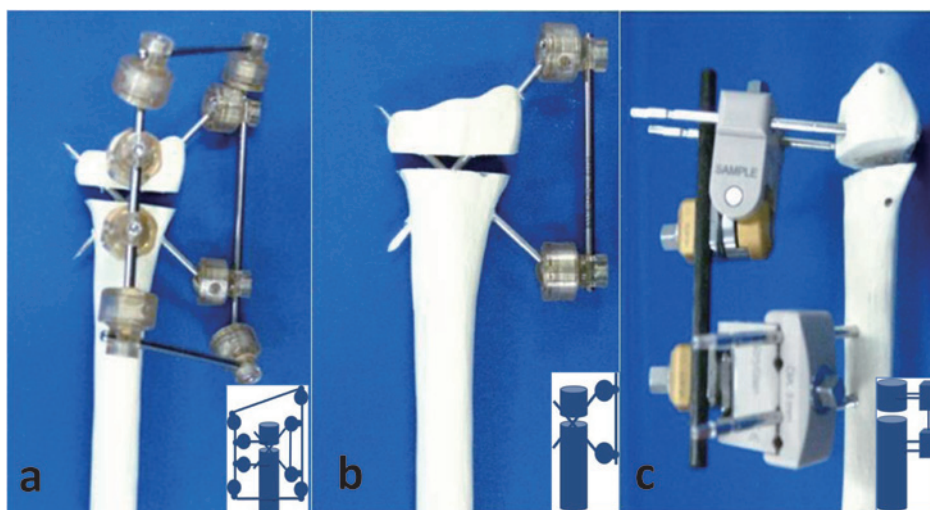


**Fig. 1(a, b, c, d, e)** Mechanism of the PinFix. The PinFix is a plastic ball-joint weighing 2.9 g that can connect pins ( $\phi 1.6$ – $2.4$  mm) with rods ( $\phi 3$  mm) at any desired angle to form a truss-structure-based fracture fixation system. ‘a’ is the view from above. ‘b’ is the view from below. ‘c’ is the view from the side. The pins are inserted across the fracture site in a crisscross fashion. This is a universal system that can construct either two- or three-dimensional fixations in any configuration. The assembly is infinite (d and e)

3 mm proximal to the styloid process. A 3-mm fracture gap was created by making a second transverse osteotomy 3 mm proximal to the initial cut, and this section of bone was removed to simulate the complete lack of cortical contact seen in severely comminuted, unstable, extra-articular, distal radius fractures. The fracture models were divided randomly into the following groups: three-dimensional PinFix fixation (Group A,  $n=5$ , Fig. 2a), two-dimensional PinFix fixation (Group B,  $n=5$ , Fig. 2b), and conventional external fixation (Hoffman II mini External Fixator Stryker, Mahwah, NJ; Group C,  $n=5$ , Fig. 2c). In the PinFix groups, 2.4-mm Kirschner wires were used for fixation. In group C, 3-mm dedicated threaded pins were used for fixation.

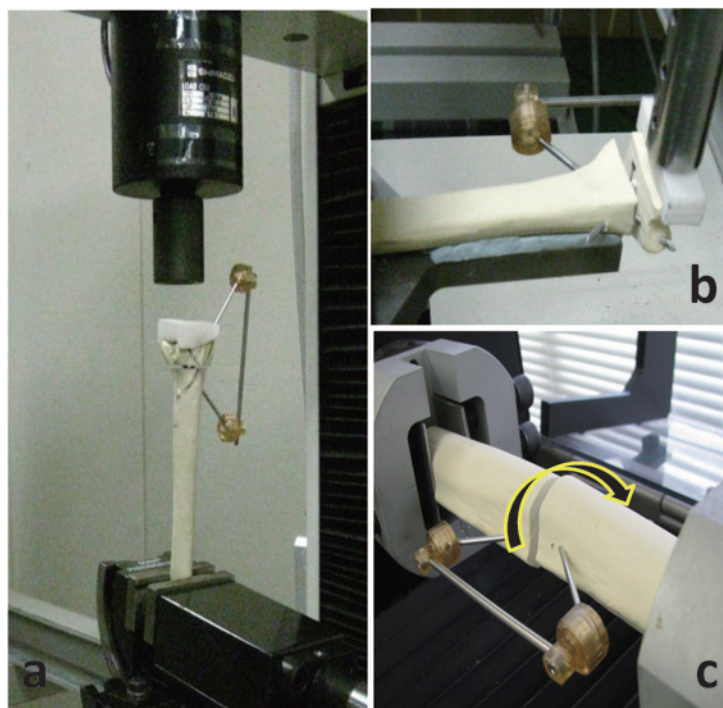
The fracture models were placed on the loading platform of a universal testing machine (Autograph AG-1, Shimazu, Kyoto, Japan). Each specimen was loaded at a rate of 1 N/s to a maximum load of 3-mm displacement for both axial (Fig. 3a) and bending loads (Fig. 3b), with 2-mm/min cross head speed. To generate optimal loading, a three-dimensional plastic gripping adaptor was put on the distal end of the sawbone radius during axial loading and on the volar side of the distal fragment during bending loading. Displacements of 1, 2, and 3 mm were compared in the three groups, and the load-displacement curve was plotted for each axial and bending load.

In the torsion loading, the machine had to grip both ends of the bones, but the radius bone could not be gripped firmly because of the shape. Instead, the femur sawbones were used. Fifteen femurs (Model #1130, Pacific Research Laboratories, Inc. Vashon, WA, USA) were prepared for torsion load testing. Both sides of the femur were cut at 10 cm from the fracture site and fixed



**Fig. 2(a, b, c)** The fracture models in mechanical loading tests

The fracture models were divided randomly into the following groups: three-dimensional PinFix fixation (Group A,  $n=5$ , *a*), two-dimensional PinFix fixation (Group B,  $n=5$ , *b*), and conventional three-dimensional external fixation (Hoffman II mini External Fixator Stryker, Mahwah, NJ; Group C,  $n=5$ , *c*).



**Fig. 3(a, b, c)** Axial, bending, and torsion loading in mechanical loading tests

Each specimen was loaded at a rate of 1 N/s to a maximum load of 3-mm displacement for both axial (*a*) and bending load (*b*), with 2-mm/min cross head speed. Torsion loading was applied as shown in '*c*'. Each specimen was loaded at a rate of 1 N/s to a maximum load of 25 N with a 2-cm moment arm in torsion.

using a metallic clamp of a testing machine (Low Capacity Torsion Testing Systems, Instron Japan, Kanagawa, Japan). The fracture fixation method using each fixator was tested in the same manner as the axial and bending loading. Torsion loading was applied as shown in Fig. 3c. Each specimen was loaded at a rate of 1 N/s to a maximum load of 25 N with a 2-cm moment arm in torsion. Rotations of 10 and 20 degrees were compared in the three groups, and the load-displacement curve was plotted.

#### *Finite element analysis*

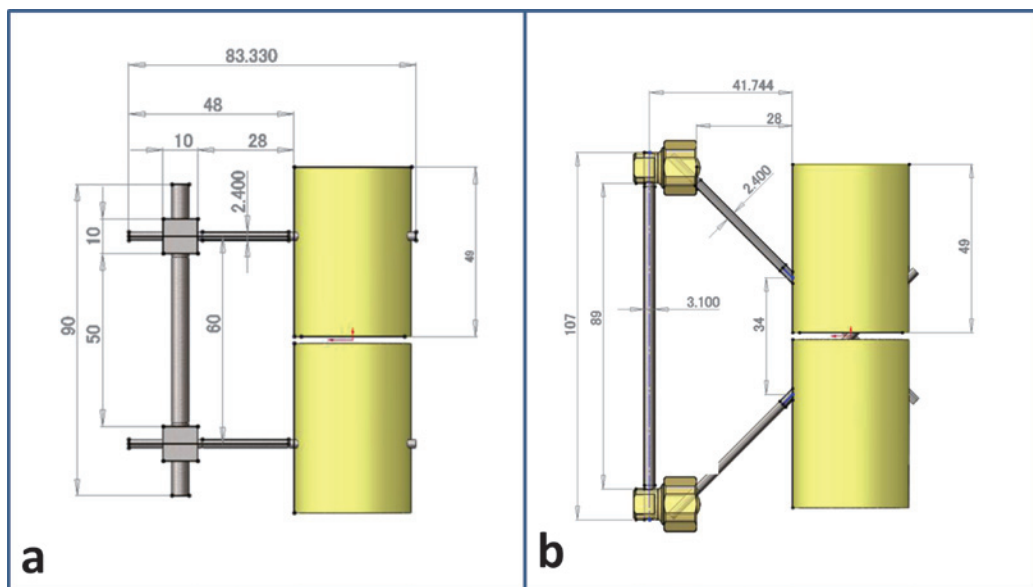
Simulated cylindrical bone that consisted of cancellous and cortical bone was created on the computer (SolidWorks Simulation, Dassault Systemes SolidWorks Corp. Waltham, MA, USA). The material properties of cortical and cancellous bone were determined based on a previous report.<sup>23)</sup> The cylinder was divided into two parts to simulate the fracture. In order to fix the fracture, cantilever-frame-fixation (C.F.F) (Fig. 4a) or truss-frame-fixation (T.F.F) (Fig. 4b) was performed. The lower end of the cylinder was set to the cornerstone, and the other end received the load. The structures of the C.F.F were constructed with rods, connectors, and pins. All of them were made of Ti-6Al-4V. In contrast, the T.F.F was constructed with rods, connectors, and pins. The materials of each part were Ti-6Al-4V, PPSU, and SUS304, respectively.

The material properties in each construct are shown in Table 1.

Axial, bending, and torsion loading were applied. Fig. 5a and 5b show the each direction of the load to C.F.F and T.F.F. Axial load stress testing (100 N), bending load stress testing (100 N), and torsional load stress testing (2 N·m) were performed. The stress of each part was recorded.

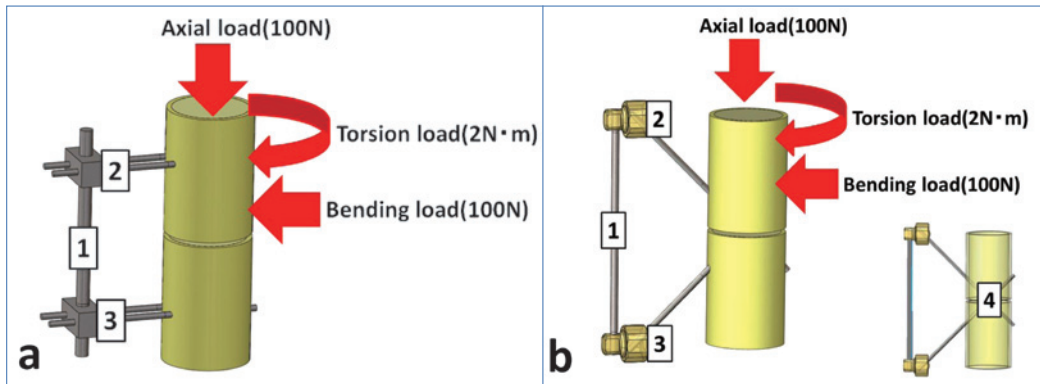
#### *Statistical analysis*

All values from the mechanical testing are expressed as means±standard deviation. Data were compared among the three groups using one-way analysis of variance (Excel 2010 statistics).



**Fig. 4(a, b)** Cantilever-frame-fixation (C.F.F. *a*) and truss-frame-fixation (T.F.F. *b*). The cylinder was divided into two parts to simulate the fracture. In order to fix the fracture, C.F.F. or T.F.F. was performed.

The level of significance for all tests was set at  $p < 0.05$ .



**Fig. 5(a, b)** Direction of the load.

'a' and 'b' show the direction of the load to C.F.F. and T.F.F. Axial load stress testing (100 N), bending load stress testing (100 N), and torsional load stress testing (2 N·m) were performed.

**Table 1** Material properties of each part in a finite element analysis

Fixation method	Parts	Material	Modulus of elasticity (N/mm <sup>2</sup> )	Poisson's ratio	Mass density (kg/m <sup>3</sup> )	Tensile strength (N/mm <sup>2</sup> )	Yield strength (N/mm <sup>2</sup> )
Truss-frame-fixation (T.F.F.: PinFix)	Connector	PPSU	2350	0.3	1290	70	-
	φ2.4mmK-wire	SUS304	190000	0.29	8000	517.017	206.807
	φ3.1mm rod	Ti-6Al-4V	110000	0.3	4430	860	760
Cantilever-frame-fixation (C.F.F.: Conventional Fixator)	Connector	Ti-6Al-4V	110000	0.3	4430	860	760
	φ2.4mm Pin						
	φ5mm rod						
-	Bone	Human bone (cortical)	17200	0.3	1640	106	-
-	Bone	Human bone (cancellous)	350	0.3	1020	7	-

**Table 2** Load applied to each portion

		Maximum stress (N/mm <sup>2</sup> )					
		Axial		Bending		Torsion	
		<i>C.F.F.</i>	<i>T.F.F.</i>	<i>C.F.F.</i>	<i>T.F.F.</i>	<i>C.F.F.</i>	<i>T.F.F.</i>
1	Rod	436.0	2.9	419.6	59.0	152.2	37.8
2	Distal connector	823.8	4.4	90.5	40.4	119.6	57.8
3	Proximal connector	848.4	4.4	1061.0	35.3	109.0	57.9
4	Pin-crossing-part		51.0		731.1		597.3

## RESULTS

*Mechanical loading tests**Axial loading:*

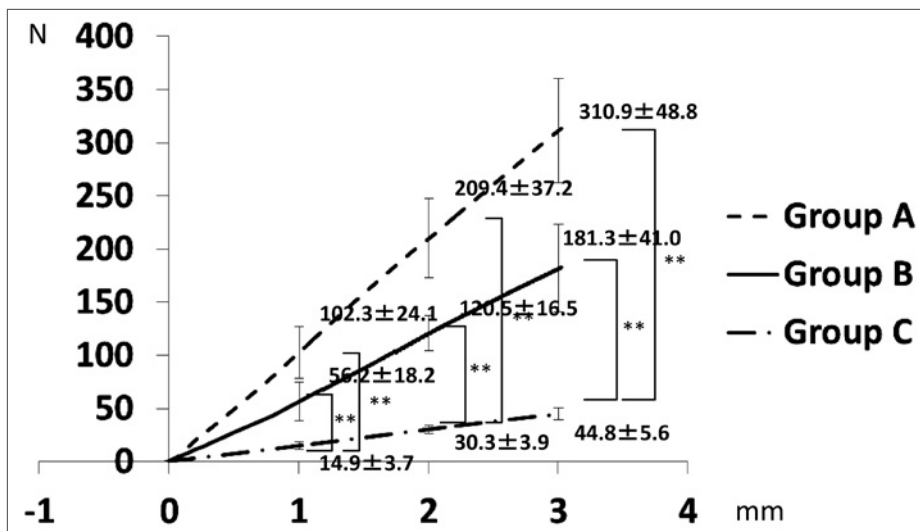
The load-displacement curve of the axial load is shown in Fig. 6. The load of 1-mm displacement for each group (A, B, C) with axial loading was  $102.3 \pm 24.1$  N,  $56.2 \pm 18.2$  N, and  $14.9 \pm 3.7$  N, respectively. That of 2-mm displacement was  $209.4 \pm 37.2$  N,  $120.5 \pm 16.5$  N, and  $30.3 \pm 3.9$  N, respectively, and 3-mm displacement was  $310.9 \pm 48.8$  N,  $181.3 \pm 41.0$  N, and  $44.8 \pm 5.6$  N, respectively.

Comparisons between Group A and C, and between Group B and C, at 1-mm, at 2-mm, and at 3-mm displacements showed significant differences, with p-values of  $p=0.0001$  and  $p=0.0134$  at 1 mm,  $p=0.0001$  and  $p=0.0111$  at 2 mm, and  $p=0.0001$  and  $p=0.0093$  at 3 mm, respectively.

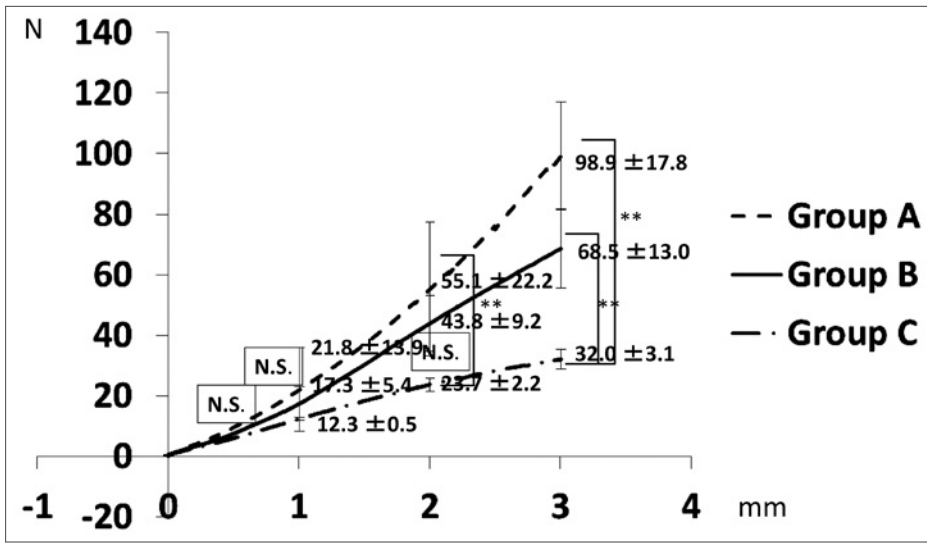
*Bending loading*

The load-displacement curve of the bending load is shown in Fig. 7. The load of 1-mm displacement for each group (A, B, C) in the bending loading was  $21.8 \pm 13.9$  N,  $17.3 \pm 5.4$  N, and  $12.3 \pm 0.5$  N, respectively. That of 2-mm displacement was  $55.1 \pm 22.2$  N,  $43.8 \pm 9.2$  N, and  $23.7 \pm 2.2$  N, respectively, and 3-mm displacement was  $98.9 \pm 17.8$  N,  $68.5 \pm 13.0$  N, and  $32.0 \pm 3.1$  N, respectively.

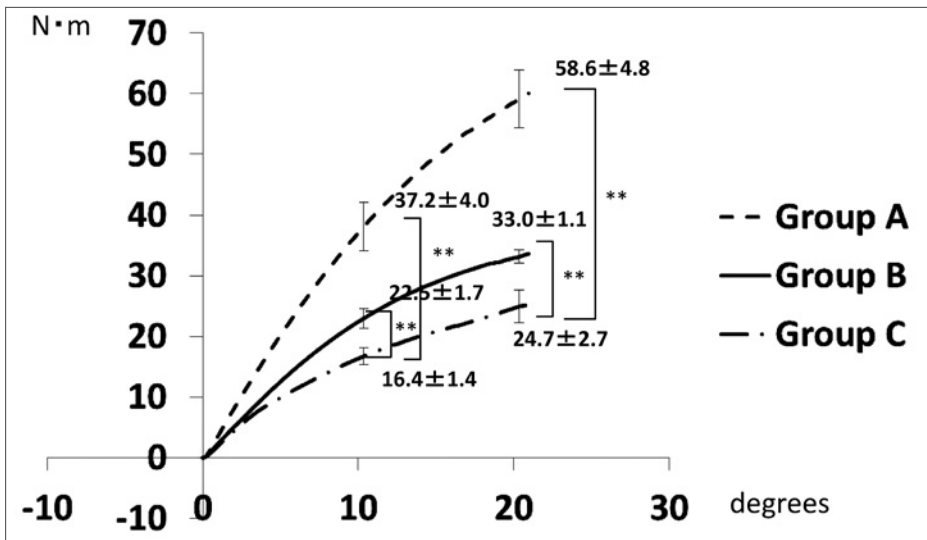
Comparisons between Group A and C and between Group B and C at 1-mm and 2-mm displacements did not show significant differences ( $p=0.2022$ ,  $p=0.4667$ , respectively). In contrast, significant differences were found at 3-mm displacement, with p-values of  $p=0.0001$  and  $p=0.0061$ , respectively.

*Torsion loading:*

**Fig. 6** The load-displacement curves of the axial load. The meaning of each line is as follows: Group A (dotted line), Group B (continuous line), and Group C (long dashed dotted line). Comparisons of two different fixations for Group A and Group C and for Group B and Group C, demonstrate significant differences at 1-mm displacement ( $p=0.0001$ ,  $p=0.0134$ , respectively), at 2-mm displacement ( $p=0.0001$ ,  $p=0.0111$ ), and at 3-mm displacement ( $p=0.0001$ ,  $p=0.0093$ ).



**Fig. 7** The load-displacement curves of the bending load. Comparisons of two different fixations for Group A and Group C and for Group B and Group C, show no significant differences at 1-mm displacement ( $p=0.2022$ ,  $p=0.4667$ , respectively). Although there are no significant differences between PinFix and conventional fixators at 1-mm and 2-mm displacements, a significant difference is seen at 3-mm displacement between Group A and Group C ( $p=0.0001$ ) and between Group B and Group C ( $p=0.0061$ ).



**Fig. 8** The load-angle curves of the torsion load. Comparisons of two different fixations for Group A and Group C and for Group B and Group C, demonstrate significant differences at 10 degrees rotation ( $p=0.0000$ ,  $p=0.0012$ ), at 20 degrees rotation ( $p=0.0001$ ,  $p=0.0111$ ), and at 3-mm displacement ( $p=0.0000$ ,  $p=0.0013$ ). The three-dimensional PinFix fixation is strongest in torsion loading.



The load-angle curve of torsion loading is shown in Fig. 8. The torque of 10-degree rotation for each group (A, B, C) in the torsion loading was  $37.2\pm 4.0$  N·m,  $22.5\pm 1.7$  N·m, and  $16.4\pm 1.4$  N·m, respectively. That of 20-degree rotation was  $58.6\pm 4.8$  N·m,  $33.0\pm 1.1$  N·m, and  $24.7\pm 2.7$  N·m, respectively.

Comparisons between Group A and C and between Group B and at 10-degree and 20-degree rotations demonstrated significant differences, with p-values of  $p<0.0000$  and  $p=0.0012$  at 10 degrees, and  $p=0.0001$  and  $p=0.0111$  at 20 degrees, respectively. The three-dimensional PinFix fixation was the strongest in torsion loading.

#### *Finite element analysis*

As can be seen in Tables 1 and 2, finite element analysis (FEA) showed remarkable differences in the stress distribution pattern between C.F.F and T.F.F. under all three loading conditions. In axial loading, stress in the rod was  $436.0$  N/mm<sup>2</sup> in C.F.F, while that in T.F.F, was  $2.9$  N/mm<sup>2</sup>, and stress at the distal and proximal connectors of C.F.F was  $823.8$  N/mm<sup>2</sup> and  $848.4$  N/mm<sup>2</sup>, while stress at the distal and proximal connectors of T.F.F. were  $4.4$  N/mm<sup>2</sup> and  $4.4$  N/mm<sup>2</sup>, respectively. In bending loading, stress on the rod was  $419.6$  N/mm<sup>2</sup> in C.F.F, while that in T.F.F was  $59.0$  N/mm<sup>2</sup>, and stress at the distal and proximal connectors of C.F.F was  $90.5$  N/mm<sup>2</sup> and  $1061.0$  N/mm<sup>2</sup>, respectively, while stress at the distal and proximal connectors of T.F.F. was  $40.4$  N/mm<sup>2</sup> and  $35.3$  N/mm<sup>2</sup>, respectively. In torsion loading, stress on the rod was  $152.2$  N/mm<sup>2</sup> in C.F.F, while that in T.F.F was  $37.8$  N/mm<sup>2</sup>, and stress at the distal and proximal connectors of C.F.F was  $119.6$  N/mm<sup>2</sup> and  $109.0$  N/mm<sup>2</sup>, respectively, while stress at the distal and proximal connectors of C.F.F was  $57.8$  N/mm<sup>2</sup> and  $57.9$  N/mm<sup>2</sup>, respectively.

The results clearly demonstrated the differences between C.F.F. and T.F.F. under all three loading conditions. On the other hand, in T.F.F., a relatively high stress concentration takes place along the crossing pins inside the bone, with values of  $51.0$  N/mm<sup>2</sup> in axial loading,  $731.1$  N/mm<sup>2</sup> in bending loading, and  $597.3$  N/mm<sup>2</sup> in torsion loading. It appears that higher stress concentrations occur in rods and connectors in C.F.F, while the same happens in crossing pins inside the bone in T.F.F. These results indicate that the applied load is converted to a pure compression or tension load along the pins in the PinFix, thereby significantly reducing stress in members outside the bones.

## DISCUSSION

Truss-frames are composed of triangles that are the simplest geometric figure that will not deform once the lengths of the sides are fixed. In comparison, a four-sided figure such as a cantilever-frame will change shape in response to external forces. Therefore, both the angles and the lengths must be firmly fixed to retain its shape.

Finite element analysis clearly showed high stress concentrations at the angles of the cantilever frame. In contrast, in the case of a truss frame, the mechanical stress is spread along the pins crossing the fracture site, and, therefore, less stress concentration occurs at the joints or along the rods. Therefore, the structure can be constructed using relatively small and weak connecting materials. Because of this, it was possible to make the PinFix with plastic parts as light as 3 g each. Despite the lightweight material used, all mechanical load testing clearly demonstrated that the PinFix truss fixation can better withstand mechanical stresses in all directions than can conventional external fixator systems.

In truss structure, it is noteworthy that the predicted axial force is significantly smaller than the bending or torsion force. This result could be explained by the law of the lever.<sup>24)</sup> In the

case of axial load analysis, the axis of the applied load is almost collinear with the longitudinal axis of the bone. Therefore, the lever arm is practically zero. In contrast, in the case of bending and torsional load testing, the axis of the load is distant from the action point. A much larger lever arm in these tests results in much higher mechanical stress at the action point.

On the other hand, FEA showed a relatively high stress concentration along the pins around the cross part despite the fact that the two pins are not connected to each other within the bone in the PinFix. This reflects the physical characteristics of the truss, in which the applied deforming force is converted into pure compression or tension stress along the parts. This type of structure is widely used in a variety of truss constructions, such as Brown truss bridges.<sup>25, 26)</sup>

The mechanical loading test clearly showed the greater stiffness of the truss-frame, both of the three- and two-dimensional PinFix, than that of the cantilever-frame. In addition, the three-dimensional PinFix was obviously stiffer than the two-dimensional PinFix. This indicates that a more robust structure can be constructed by combining the simple truss structures. In fact, according to Pouangare,<sup>27)</sup> a complex three-dimensional truss, alias 'the space-truss', can give constructions with extremely high strength.

In our mechanical study, sawbones were used. The main advantage<sup>28)</sup> of using sawbones is that their use has been well validated in comparisons with cadaver specimens, and they are considerably better represented in hand and upper extremity biomechanics research. In fact, a variety of problems with cadaver specimens have been pointed out, including high cost, tenuous availability, handling and storage challenges, and a remarkable degree of inter-specimen variability that reportedly exceeds 100% of the mean in some metrics.

Historically, crisscross pin fixation has been widely used in fracture management.<sup>13)</sup> It is a less invasive fracture fixation technique that can be performed at any medical facility that uses simple fracture treatment devices such as image intensifiers, drills, and Kirschner wires. In addition, the technique is also widely used during surgery to temporarily maintain reduction until internal fixation with plates is completed. The caveat is that it is much less reliable compared to other fracture techniques and almost always requires additional supports such as cast immobilization.<sup>29)</sup> However, once the PinFix is attached to the Kirschner wires, it instantaneously becomes a stronger supporting device compared to other fixation systems.

PinFix is not the only external fixator using the truss structure. The CPX system developed by Mirza *et al.* is a uniplane external fixation system that supports multiple, small, 1.6-mm cross-pins.<sup>30)</sup> Using a cadaveric fracture model, Strauss *et al.* compared the CPX system with volar locking plate fixation and concluded that there was no significant difference between the two fixation techniques. Their results also proved the mechanical advantages of a truss system. The problem of CPX is that it is a site-specific fracture fixation system. It can only be applied to a limited number of fracture types of the distal radius, such as AO type B2 or B3.<sup>31)</sup> In contrast, the PinFix can be used in various types and at various sites of fracture without requiring any special devices. Indeed, a distal radius fracture might be a good indication for the PinFix, and it can also be used for forearm, elbow, and humerus fractures.

This study clearly demonstrated the usefulness of introducing the basics of structural engineering to the designing of fracture fixation devices.

In conclusion, by taking the mechanical advantages of truss structures into consideration, a simple fracture fixation device, the PinFix, which can convert a simple cross pin fixation into an extremely robust external fixation system by inducing drastic changes in load distribution, was successfully developed.

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